

# Undercarriage Design of Excavator Model in Application of Various Track Drive

Nazaruddin,<sup>a</sup> and Kiki<sup>a</sup>, Gunawan<sup>a</sup>

<sup>a</sup>) Laboratorium Hidrolik dan Pneumatik, Jurusan Teknik Mesin, Fakultas Teknik Universitas Riau

\*Corresponding author: nazaruddin.unri@yahoo.com

## Paper History

Received: 11-November-2015

Received in revised form: 30-November-2015

Accepted: 30-December-2015

## ABSTRACT

The excavator is the units of the heavy equipment that serves the physical development sectors such as mining excavations in the area, establishing or expanding roads and expand agricultural land and other physical development . One part of the excavator that has a very large role in the undercarriage . Undercarriage is a component of the heavy equipment that serves as a driver and has a track drive right and left track drive . This research was conducted with the aim to make modeling as excavators in general by using materials available in the market. Next, calculate the speed , direction of turn with a different variation of the track (ceramic , asphalt , soil) and the maximum tilt angle that can be achieved by the excavator . From the test data and calculations have been carried out with 3 times the gear reduction is obtained without load speed excavator bucket is 0.25 m/s while using a load of 1.5 kg bucket excavator speed is 0.24 m/s at the track ceramics . While the direction of maximum inflection occurs on the track with a diameter ceramic to turn right for 995 mm and 782 mm turn left. At the maximum angle of incline can be obtained at 10 degrees .

**KEY WORDS:** *Undercarriage; Track Drive; Excavator*

## 1.0 INTRODUCTION

Excavator is one of heavy equipment to carry out the various of construction such highway, canal, agriculture and mining project. The using of the excavator is to accelerate the construction

process and save the consuming time in construction and operation process in order to reduce cost. In this study, the effort to obtain the optimum heavy equipment is continuously done in order to maximize the operation of excavator by modification of undercarriage part[1]. As shown in figure 1.1

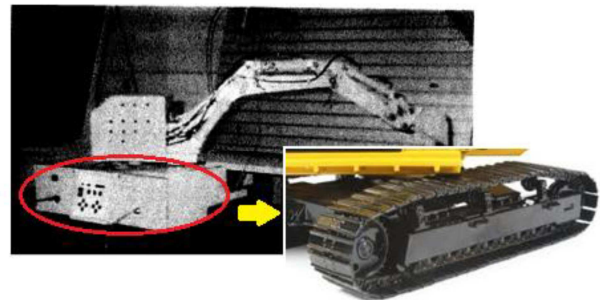


Figure 1.1 Excavator models that will be modified[1]

## 2.0 THEORY

Undercarriage is essential part of excavator which consists of several component to support the movement of excavator such as sprocket, final drive unit, track shoe, track link, track frame, track roller, front idler as shown in figure 2.2,

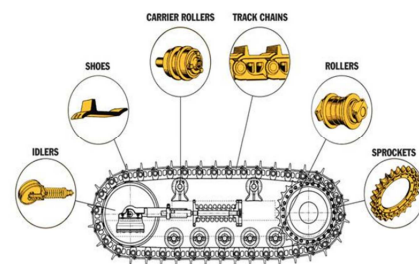


Figure 2.2 Undercarriage Component [4]



The dimensions of the undercarriage were taken from 1:14 scaled model of the actual size so structured such that generates the layout as in Figure 3.2 below

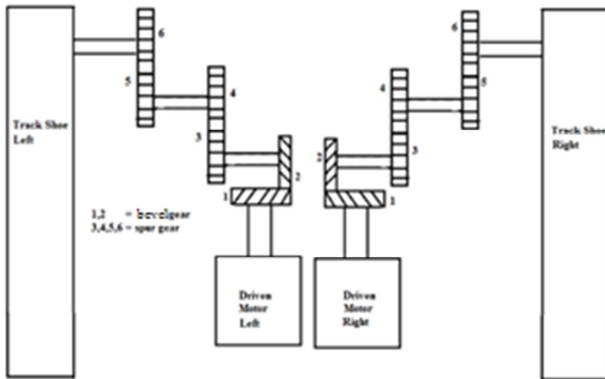


Figure 3.2 Layout of Power Transmission of Undercarriage

Power transmission from driven motor to each track shoe using a pair of bevel gear (gear 1 and gear 2) and two pairs of spur gears (pinion gears 3 to 6).

The layout made its CAD drawings into 3D models as shown in Figure 3.3 below

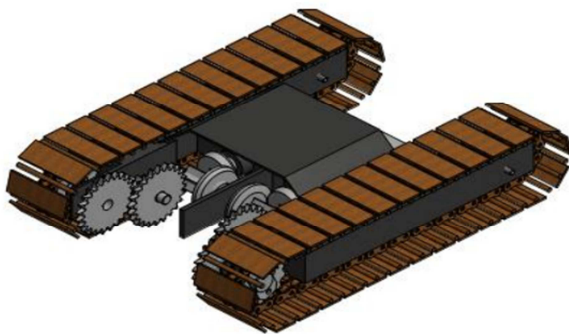


Figure 3.3 Result of 3D Design of Undercarriage

The transmission arrangement to obtain the undercarriage low speed of 3.6 km/h and undercarriage high speed 5.5 km/ as the speed of the excavator is generally where a motor driven rotation of 2000 rpm

### 3.2 Undercarriage Fabrication

The process of making undercarriage conducted at the Laboratory of Production of Mechanical Engineering, University of Riau. Flow chart of the manufacturing process can be seen in Figure 3.5.

Making the frame by means of Steel box cut using grinding pieces with a length of 30 cm. Then one end of the steel box section is cut 10 cm with the aim to include idler and tension to set the track shoe and make the position of bearing needed to simplify and expedite the undercarriage in the running

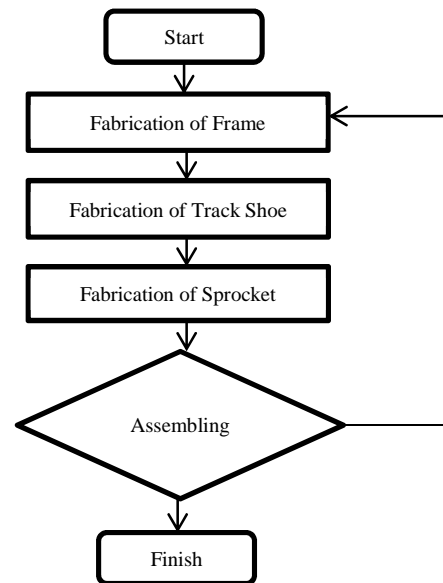


Figure 3.4 Flowchart of Fabrication

Track shoe is made by cutting a plate that has a thickness of 2 mm and a length of 50 mm and a width of 15 mm. Plates that have been cut was installed in the cracks of the iron cylinder has been connected with the chain sprocket. Furthermore, the merger between the chain sprocket or the so-called track shoe with a steel box that has been cut. The result as in Fig. 3.5



Figure 3.5 Assembling of Track Shoe

Installation of the sprockets and gears. After completion of the formation of the frame, the next step is the process of turning to make the position of the gear and then making a connection between the position of the gear to the frame (see in Fig. 3.6). The gears used are gears that are easy to obtain, is composed of two types of gears is a hypoid gear and spur gear and performed 3 times to get a final speed reducer such as undercarriage in general.



(a) The position of the gear (b) Sprocket and gear on the frame

Figure 3.6 Installation Stand Gear and Sprocket on Frame

### 3.3 Undercarriage Testing

At this stage we will perform an undercarriage testing activities. This process can be seen in Fig. 3.7 following a flow chart of the testing process.

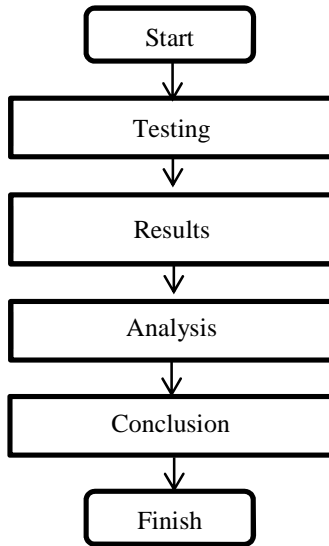


Figure 3.7 Flowchart of Undercarriage Testing

This test was conducted to determine the outcome of designing and manufacturing has been done. Testing is done with three types of tests, among others:

1. Testing speed and a final round of undercarriage  
This test was conducted to determine the speed and the final round of the excavator. Testing is done by giving the mileage on the excavator which was then measured how long it takes the excavator. And rotation speed testing is done with 3 variations of the track are: ceramics, asphalt and soil. While the rotation speed is obtained from then converted into angular velocity rounds finally obtained value.
2. Testing the direction of turn undercarriage  
This test is done to determine how much deflection angle that can be done by the excavator. Testing is also done with a variation of 3 tracks namely: ceramics, asphalt and soil (see in Fig. 3.8)

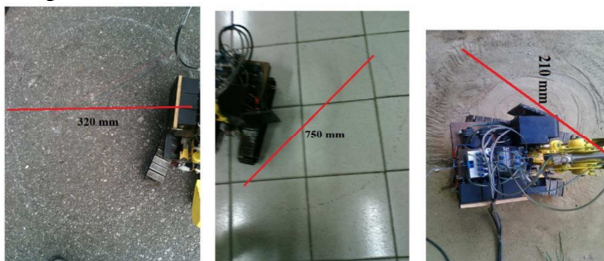


Figure 3.8 Testing Direction Turn

3. Tests on inclined plane.  
This test is done to determine how much the maximum angle

that can be taken by the excavator. Testing was done with 6 variation of the angle that is 5°, 10°, 15°, 20° and 25° (see in Fig. 3.9)



Figure 3.9 Comparison Testing on inclined plane

### 4.0 RESULT AND DISCUSSION

The testing had been conducted to obtained the data of speed and excavator's manouver in several models of track like ceramic, asphalt and land.

#### In Asphalt Track

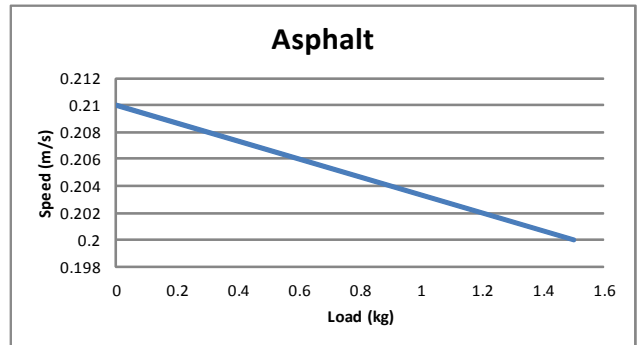


Figure 4.1 Load vs Speed in Asphalt Track

In the Figure 4.1 show the result of undercarriage experiment in the asphalt track, where the increasing of loads cause the speed of excavator decreased. Figure 4.2 illustrate the load vs rotation as well, the increasing of load affect to the rotation of undercarriage final drive.

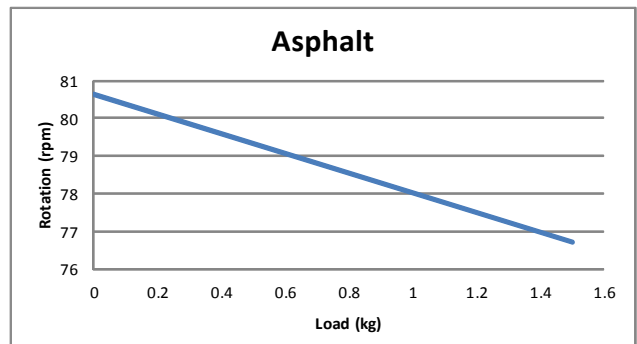


Figure 4.2 Load vs Rotation in Asphalt Track

In Ceramic Track

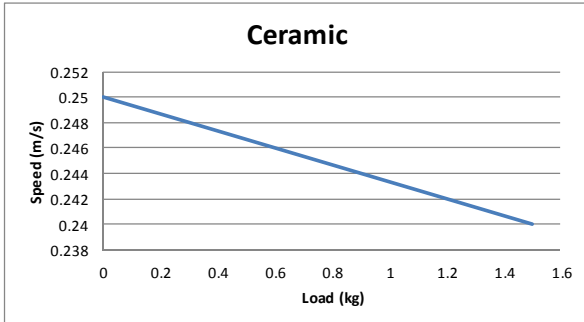


Figure 4.3 Load vs Speed in Ceramic Track

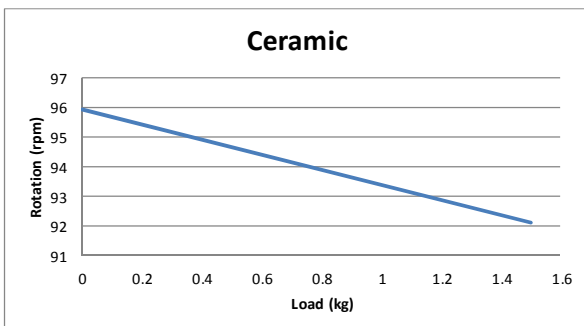


Figure 4.4 Load vs Rotation In Ceramic Track

In the ceramic track as shown in figure 4.3 shows the speed of excavator against the increasing of load, where the speed linearly decrease. The figure 4.4 shows the rotation of undercarriage decrease by increasing of load.

In Unpaved Road

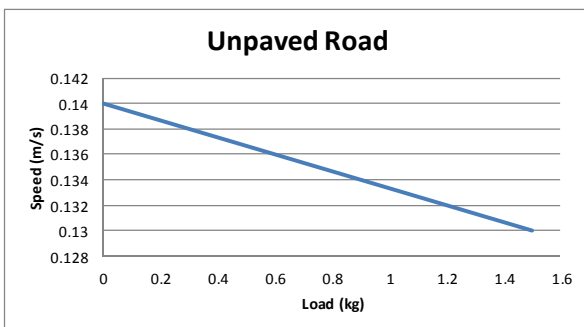


Figure 4.5 Load vs Speed in Unpaved Road

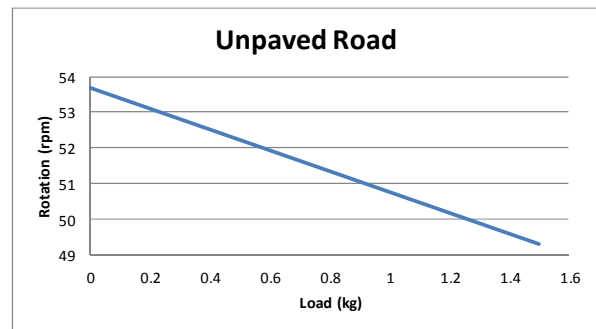


Figure 4.6 Load vs Rotation in Unpaved Road

In the figure 4.5 show the comparison between load and speed in unpaved road. It indicates the speed of excavator on unpaved road track is the lowest speed when compare to other track, as well as the rotation of undercarriage final drive become decreasing by increasing of load.

5.0 CONCLUSION

From the results of the design, manufacture and calculations have been performed, then obtained some conclusions of a thesis entitled designing and manufacturing excavator undercarriage components on the model in the Laboratory of Hydraulic and Pneumatic University of Riau, namely:

1. Provided the form of the undercarriage of the design and manufacturing has been done with the long dimension of undercarriage 331 mm, width 234 mm and height 78.6 mm
2. From the test results that have been done so by doing three times the reduction of the gear system and using the motor power windows Toyota gained speed as the excavator generally is 3.6 km / h or 1 m / s for low speed undercarriage and 5 , 5 km / h or 1.5 m / s for high speed undercarriage.
3. From the examination of the direction of turn can do the undercarriage on the track asphalt, ceramics and soil the minimum diameter are on the track of land with a diameter of 210 mm.
4. The maximum slope that can be achieved excavator which is 10 degrees with a power of 83.60 Watts.
5. Obtained a modified cable remote control to move the undercarriage..

REFERENCE

1. Julianto and Nazaruddin, 2004, Perencanaan dan Pengujian Model *Excavator*, Kertas Karya Prodi D3 Teknik Mesin Universitas Riau.
2. Jati, Hidayah, 2011. Peningkatan Perawatan Komponen *Undercarriage* Alat Berat. Depok: Universitas Indonesia.
3. Prasetyo, Deta. 2010. Pembuatan Alat Praktikum Perawatan Sistem Transmisi Roda Gigi. Surakarta : Universitas Sebelas Maret
4. Soleh, Muhammad. 2005. Sistem Operasi dan Perawatan *Travel Motor* pada *Excavator* Hitachi Zaxis-200, Prodi D3 Teknik Mesin Universitas Riau.

5. Shigley, 2006, Mechanical Engineering Design 8<sup>th</sup>, McGraw-Hill Companies, USA